



Fluid Membranes I:

Geometry

Variational Principles

Stress and Conservation Laws

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Free Fluid Membranes

- Striking feature of cell membranes:
behave as fluid membranes on mesoscopic length scales
- Mechanical properties in equilibrium:
Largely captured by geometrical degrees of freedom of surface
Fluid \Rightarrow Shear unpenalized

Described with extraordinary accuracy by Helfrich bending energy
– or its extensions
- Quadratic in curvature \Rightarrow non-linear field theory:
$$\nabla^2 \text{Curvature} + \text{Curvature}^3 = \text{Sources}$$
- nevermind physics, just getting geometry right is no mean feat

All non-trivial geometries are stressed

- Shape in most interesting bio-membranes controlled by external forces
 - both global and local

They themselves play a role in mediating interactions

- Forces are transmitted by the stress established in the membrane
- These stresses are completely determined by the geometry
 - unusual in a soft matter system

Thus by examining the geometry it becomes possible to determine the forces that are the source of stress

- In this lecture I will show how to identify this stress tensor and examine a few of its properties

A little geometry

- Parametrized closed surface: $(u^1, u^2) \rightarrow \mathbf{X}(u^1, u^2)$

$$\mathbf{X} = (X^1, X^2, X^3) \quad 3 \text{ functions of 2 variables}$$

- (1) Tangents $\mathbf{e}_a = \partial_a \mathbf{X}$, $a = 1, 2$
(2) Unit normal \mathbf{n}

- Two surface tensors encode geometry:

(1) Induced metric: $g_{ab} = \mathbf{e}_a \cdot \mathbf{e}_b$

(2) Extrinsic curvature: how fast \mathbf{n} rotates along parameter curves

Weingarten equation: $\partial_a \mathbf{n} = K_a^b \mathbf{e}_b$

$$K_{ab} = \mathbf{e}_a \cdot \partial_b \mathbf{n} = g_{ac} K_b^c, \quad K_{ab} = K_{ba}$$

- Monge: $\mathbf{r} \rightarrow (\mathbf{r}, h(\mathbf{r}))$

$$K_{ab} = -\partial_a \partial_b h / (1 + |\nabla h|^2)^{3/2} \approx -\partial_a \partial_b h \text{ w.r.t tangent plane}$$

Gauss equations

- Gauss : a statement about tangent vector change

$$\partial_a \mathbf{e}_b = \Gamma_{ab}^c \mathbf{e}_c - K_{ab} \mathbf{n}.$$

Same K_{ab} !; $\Gamma_{ab}^c = \Gamma_{ba}^c$

- Identify Γ_{ab}^c as Christoffel symbols associated with the surface covariant derivative ∇_a :

$$\begin{aligned} \Gamma_{ab}^c &= g^{cd} \mathbf{e}_d \cdot \partial_a \mathbf{e}_b \\ &= \frac{1}{2} g^{cd} (\partial_a g_{bd} + \partial_b g_{ad} - \partial_d g_{ab}). \end{aligned}$$

Follows directly from the definition $g_{ab} = \mathbf{e}_a \cdot \mathbf{e}_b$.

- Introduce covariant derivative $\nabla_a \mathbf{e}_b = \partial_a \mathbf{e}_b - \Gamma_{ab}^c \mathbf{e}_c$
Write Gauss as $\nabla_a \mathbf{e}_b = -K_{ab} \mathbf{n}$.

Why bother: $\nabla_a \mathbf{e}_b$, unlike $\partial_a \mathbf{e}_b$ transforms as a second rank covariant tensor under reparametrization.

Helfrich Hamiltonian

- Bending energy is quadratic in curvature

Eigenvalues: $K^a_b V_I^b = C_I V_I^a \quad I = 1, 2$

Two symmetric curvature scalars:

(1) $K := g^{ab} K_{ab} = C_1 + C_2$

(2) $K_G = \det K^a_b = \frac{1}{2}(K^2 - K^{ab} K_{ab}) = C_1 C_2$

(3) Third degree of freedom is a direction

- Helfrich bending energy (ignore spontaneous curvature):

$$H_0[\mathbf{X}] = \frac{1}{2} \kappa \int dA (C_1 + C_2)^2$$

- Topological Gaussian energy: $H_{GB} = \bar{\kappa} \int dA C_1 C_2$
will set boundary conditions!

- Third quadratic: $(C_1 - C_2)^2 = (C_1 + C_2)^2 - 4C_1C_2$

$$H = \frac{1}{2} \int dA (C_1 - C_2)^2$$

From bulk point of view H_0 unique!

- Area, enclosed volume, etc fixed using global Lagrange multipliers

Variational Principles I

- $H[\mathbf{X}]$ a functional of \mathbf{X}

$$A = \int d^2u |\mathbf{e}_1 \times \mathbf{e}_2| = \int d^2u \sqrt{g}$$

$$\text{Lagrange identity: } |\mathbf{e}_1 \times \mathbf{e}_2|^2 = \det g_{ab} = g$$

$$\text{More generally } H = \int dA \mathcal{H}[g_{ab}, K_{ab}]$$

- Response to a small deformation of the surface?

$$\mathbf{X}(u^1, u^2) \rightarrow \mathbf{X}(u^1, u^2) + \delta\mathbf{X}(u^1, u^2)$$

$$\text{Decompose: } \delta\mathbf{X} = \Phi^a \mathbf{e}_a + \Phi \mathbf{n}$$

$$\text{Tangential part } \delta_{||}\mathbf{X} = \Phi^a \mathbf{e}_a = \Phi^a \partial_a \mathbf{X}$$

– Behavior of scalar under change of parameter: $u^a \rightarrow u^a - \Phi^a(u^1, u^2)$

– g_{ab} and K_{ab} respond accordingly

In absense of additional material degrees of freedom: only observable effect is boundary displacement

- Tangential deformation of H

$$\delta_{\parallel} \int dA \mathcal{H} = \int dA \nabla_a (\mathcal{H} \Phi^a)$$

Need only $\delta_{\parallel} \mathcal{H} = \Phi^a \partial_a \mathcal{H}$

$\delta_{\parallel} g_{ab} = \nabla_a \Phi_b + \nabla_b \Phi_a$ so that $\delta_{\parallel} \sqrt{g} = \sqrt{g} \nabla_a \Phi^a$

- Divergence theorem: $\int dA \nabla_a V^a = \int ds l_a V^a$

l_a is unit normal to the boundary out of the surface: $l_a V^a = \mathbf{l} \cdot \mathbf{V}$

$\delta_{\parallel} H = 0$ on closed surface; no role in bulk defos, equilibrium or not!

- $\delta_{\parallel}^2 H$ not so simple

Bianchi-type Identities

- $H = \int dA \mathcal{H}[g_{ab}, K_{ab}]$:

$$T^{ab} = -2\delta H / \delta g_{ab} / \sqrt{g}$$

$$H^{ab} = \partial \mathcal{H} / \partial K_{ab}$$

- Invariance under reparam: $\Leftrightarrow \nabla_a f^{ab} + K^{ab} f_a = 0$

$$f^{ab} = T^{ab} - H^{ac} K_c^b$$

$$f^a = -\nabla_b H^{ab}$$

Analogue of Bianchi identity for gravity: T^{ab} is not the full story!
Independent of equilibrium!

But not contentless in general ...

- Add material fields: say vector field v^a , $\mathcal{H}[g_{ab}, K_{ab}, v^a]$

$$\text{Euler Lagrange } \mathcal{V}_a = \delta H / \delta v^a / \sqrt{g}$$

However, in equilibrium, $\delta_\Phi v^a$ does not contribute:

$\nabla_a f^{ab} + K^{ab} f_a = 0$ represents conservation of v^a
whether or not surface is in equilibrium

If H is independent on K_{ab} : $\nabla_a T^{ab} = 0$.

Remains to identify appropriate conservation law for \mathbf{X} ?

First look at normal deformations

Normal deformations

- Saw that tangent defo does not contribute to Euler-Lagrange derivative
Normal deformation of $H[\mathbf{X}]$ can be cast as

$$\delta_{\perp} H[X] = \int dA (\mathcal{E} \Phi + \nabla_a V_{\perp}^a) .$$

\mathcal{E} is the (normal) Euler-Lagrange derivative; in equilibrium $\mathcal{E} = 0$

- To identify \mathcal{E} , need to peel off derivatives of Φ in the bulk using integration by parts. These derivatives are then collected in a divergence
– V_{\perp}^a associated vector field
- Induced normal deformations of g_{ab} and K_{ab} :
 - (1) $\delta_{\perp} g_{ab} = 2K_{ab} \Phi$
 - (2) $\delta_{\perp} K_{ab} = -(\nabla_a \nabla_b - K_{ac} K^c_b) \Phi$

Bending energy and shape equation

- $H[\mathbf{X}] = \frac{1}{2} \int dA K^2$

- Eqs.(1) & (2) on previous slide \Rightarrow

$$\delta_{\perp} \sqrt{g} = \sqrt{g} g^{ab} \delta_{\perp} g_{ab} = \sqrt{g} K \Phi; \quad \delta_{\perp} K = -\Delta \Phi - K_{ab} K^{ab} \Phi$$

$$\delta_{\perp} H = \int dA K \left[-\nabla^2 \Phi - \left(K_{ab} - \frac{1}{2} g_{ab} K \right) K^{ab} \Phi \right]$$

Integrate by parts twice \Rightarrow Euler-Lagrange:

$$\mathcal{E} = -\nabla^2 K - K \left(K_{ab} - \frac{1}{2} g_{ab} K \right) K^{ab} .$$

$\mathcal{E} = 0$ when $K = 0$ or when $K_{ab} = \frac{1}{2} g_{ab} K$, a sphere
 Area, enclosed volume constant: $\mathcal{E} + \sigma K - P = 0$

- Limitation: looks nothing like a conservation law
 Physics involves forces which set up stress which transmits the forces:
 How is this stress connected to the geometry ?

Variational Principles II: Auxiliary Variables

- H depends on \mathbf{X} only through g_{ab} and K_{ab}

Response of H to defo: $\mathbf{X} \rightarrow \mathbf{X} + \delta\mathbf{X}$ is through g_{ab} and K_{ab}

Snag: they are not independent!

Depend on \mathbf{X} through the tangent vectors $\{\mathbf{e}_a, \mathbf{n}\}$

- Structural constraints connect them to \mathbf{X} :

$$g_{ab} = \mathbf{e}_a \cdot \mathbf{e}_b, \quad K_{ab} = \mathbf{e}_a \cdot \partial_b \mathbf{n}$$

$$\mathbf{e}_a = \partial_a \mathbf{X}, \quad \mathbf{e}_a \cdot \mathbf{n} = 0, \quad \mathbf{n}^2 = 1$$

Enforce using Lagrange multipliers:

$$H_C = H[g_{ab}, K_{ab}] + \int dA \mathbf{f}^a \cdot (\mathbf{e}_a - \partial_a \mathbf{X})$$

$$+ \int dA \left[\lambda_{\perp}^a (\mathbf{e}_a \cdot \mathbf{n}) + \lambda_n (\mathbf{n}^2 - 1) \right]$$

$$+ \int dA \left[\Lambda^{ab} (K_{ab} - \mathbf{e}_a \cdot \partial_b \mathbf{n}) + \lambda^{ab} (g_{ab} - \mathbf{e}_a \cdot \mathbf{e}_b) \right]$$

Conservation laws

$$\begin{aligned}
 H_C = & H[g_{ab}, K_{ab}] + \int dA \mathbf{f}^a \cdot (\mathbf{e}_a - \partial_a \mathbf{X}) \\
 & + \int dA \left[\lambda_{\perp}^a (\mathbf{e}_a \cdot \mathbf{n}) + \lambda_n (\mathbf{n}^2 - 1) \right] \\
 & + \int dA \left[\Lambda^{ab} (K_{ab} - \mathbf{e}_a \cdot \partial_b \mathbf{n}) + \lambda^{ab} (g_{ab} - \mathbf{e}_a \cdot \mathbf{e}_b) \right]
 \end{aligned}$$

- Treat g_{ab} , K_{ab} , \mathbf{e}_a , \mathbf{n} and \mathbf{X} as independent variables
- Noether's theorem: Translational invariance of $H[\mathbf{X}] \Rightarrow$
Existence of conserved stress tensor
- \mathbf{X} only appears in the tangential constraint

$$H_C = [\text{terms independent of } \mathbf{X}] + \int dA \mathbf{f}^a \cdot (\mathbf{e}_a - \partial_a \mathbf{X})$$

$$\Rightarrow \delta_{\mathbf{X}} H_C = \nabla_a \mathbf{f}^a$$

- In Equilibrium \mathbf{f}^a is conserved: $\nabla_a \mathbf{f}^a = 0$

In general, $\mathbf{f}^a \neq 0$ even if $\nabla_a \mathbf{f}^a = 0$

- \mathbf{f}^a will be identified as the stress tensor
Transmits forces along the membrane, a 3×2 matrix

$\mathbf{f}^1 =$ force transmitted across $u^1 =$ constant

Stress depends only on the geometry

- Construction: Vary \mathbf{e}_a , \mathbf{n} , g_{ab} and K_{ab}

$$\mathbf{f}^a = (T^{ab} - H^{ac}K_c^b) \mathbf{e}_b - \nabla_b H^{ab} \mathbf{n}$$

(1) $T^{ab} \approx -2\delta H/\delta g_{ab}/\sqrt{g}$ not conserved

(2) $H^{ab} = \partial\mathcal{H}/\partial K_{ab}$

Remember: Energy density \mathcal{H} is a functional of g_{ab} and K_{ab}

- Important point:
Stress completely determined by membrane geometry

Stress in a fluid membrane:

- $\mathcal{H} = \kappa K^2/2 + \bar{\kappa} \det K^a_b + \sigma$, multiplier σ fixes $A \Rightarrow$

$\mathbf{f}^a = f^{ab} \mathbf{e}_b + f^a \mathbf{n}$ where

$$f^{ab} = \kappa K (K^{ab} - \frac{K}{2} g^{ab}) - \sigma g^{ab}, \quad f^a = -\kappa \nabla^a K$$

- f^{ab} local quadratic in K_{ab} ; non-isotropic (not a perfect fluid)
Lines of tangential stress \parallel lines of curvature \Rightarrow
Stress highs and lows correlate with curvatures

Gaussian energy does not contribute explicitly: but $H^{ab} \neq 0$
– as a result it does contribute to boundary conditions (see below)

- Normal stress requires curvature gradients
- $K = 0$ in minimal surfaces. No stress if tension is not applied

Euler-Lagrange from conservation law

- Divergence structure also mutilated in Euler-Lagrange equation
- $\nabla_a \mathbf{f}^a = P \mathbf{n}$ vs. $-2\nabla^2 K + \text{cubic in } K_{ab} = P$

Discrepancy: three conservation laws vs. one shape equation?

- Project $\nabla_a \mathbf{f}^a = P \mathbf{n} \perp$ and \parallel

$$\mathbf{f}^a = f^{ab} \mathbf{e}_b + f^a \mathbf{n}$$

$$\begin{aligned} \nabla_a f^a - K_{ab} f^{ab} &= P \\ \nabla_a f^{ab} + K^b_a f^a &= 0 \end{aligned}$$

Dismantle divergence:

(\perp) is shape equation

(\parallel) are Bianchi ids (consistency conditions)

Monge

- Simple structure mutilated in Monge gauge
- Height function, $\mathbf{r} \rightarrow h(\mathbf{r})$, small gradient $(\nabla h)^2 \ll 1$
Adapted basis $\mathbf{E}_1, \mathbf{E}_2, \mathbf{k}$

$$\mathbf{f}^i \sim (\kappa T_B^{ij} + \sigma T_M^{ij}) \mathbf{E}_j + N^i \mathbf{k}$$

$$T_B^{ij} = \nabla^2 h \nabla^i \nabla^j h - \frac{1}{2} (\nabla^2 h)^2 \delta^{ij} - \nabla^i (\nabla^2 h) \nabla^j h$$

$$T_M^{ij} = \nabla^i h \nabla^j h - \frac{1}{2} (\nabla h)^2 \delta^{ij}$$

$$N^i = \nabla^i \nabla^2 h$$

- $\nabla_i T^{ij} = 0$ when $\kappa (\nabla^2)^2 h - \sigma \nabla^2 h = 0$
 $T_B^{ij} \neq T_B^{ji}$ a gauge artifact; $\nabla_i T_{ji} \neq 0$

Boundary terms:

$$\begin{aligned}
 H_C = & H[g_{ab}, K_{ab}] + \int dA \mathbf{f}^a \cdot (\mathbf{e}_a - \partial_a \mathbf{X}) \\
 & + \int dA \left[\lambda_{\perp}^a (\mathbf{e}_a \cdot \mathbf{n}) + \lambda_n (\mathbf{n}^2 - 1) \right] \\
 & + \int dA \left[\Lambda^{ab} (K_{ab} - \mathbf{e}_a \cdot \partial_b \mathbf{n}) + \lambda^{ab} (g_{ab} - \mathbf{e}_a \cdot \mathbf{e}_b) \right]
 \end{aligned}$$

The variations wrt both \mathbf{X} and \mathbf{n} come with boundary terms:

$\mathbf{X} \rightarrow \mathbf{X} + \delta\mathbf{X}$:

$$\delta H_C = \int dA \nabla_a \mathbf{f}^a \cdot \delta\mathbf{X} - \nabla_a [\mathbf{f}^a \cdot \delta\mathbf{X}]$$

$\mathbf{n} \rightarrow \mathbf{n} + \delta\mathbf{n}$ modulo $H^{ab} = -\Lambda^{ab}$:

$$\delta H_C = \text{bla} + \int dA \nabla_a [H^{ab} \mathbf{e}_b \cdot \delta\mathbf{n}]$$

Contribution from K_G to $\mathbf{l} \cdot \delta\mathbf{n}$!

Interpretation of f^a :

Let bulk be in equilibrium outside of N boundaries/embedded particles
Construct a curve around one of them, Γ say

- Under deformation, non-vanishing in neighborhood of Γ :

$$\delta H_C = - \oint_{\Gamma} ds l_a \mathbf{f}^a \cdot \delta \mathbf{X} + \oint_{\Gamma} ds l_a H^{ab} \mathbf{e}_b \cdot \delta \mathbf{n}$$

- Translate Γ : $\delta \mathbf{X} = \delta \mathbf{a}$, $\delta H_C = -\mathbf{a} \cdot \oint_{\Gamma} ds l_a \mathbf{f}^a$
 $l_a \mathbf{f}^a$ as the force/unit length acting on Γ

- Rotate Γ : $\delta \mathbf{X} = \mathbf{b} \times \mathbf{X}$, $\delta \mathbf{n} = \delta \mathbf{b} \times \mathbf{n}$,

$$\delta H_C = -\mathbf{b} \cdot \oint_{\Gamma} ds l_a \left[\mathbf{X} \times \mathbf{f}^a + H^{ab} \mathbf{e}_b \times \mathbf{n} \right] .$$

Conserved Torque Tensor $\mathbf{m}^a = \mathbf{X} \times \mathbf{f}^a + H^{ab} \mathbf{e}_b \times \mathbf{n}$

Conformal stretching

- fix metric using local Lagrange multipliers \mathcal{T}^{ab} :

$$H_C \rightarrow H_C - \frac{1}{2} \int dA \mathcal{T}^{ab} (g_{ab} - \Omega(u) g_{ab}^{(0)})$$

$$f^{ab} \rightarrow f^{ab} + \mathcal{T}^{ab}, \quad f^a \rightarrow f^a$$

Stress no longer completely determined by geometry!

- Euler-Lagrange:

$$(\perp) \quad \mathcal{E} = 0 \rightarrow \mathcal{E} - K_{ab} \mathcal{T}^{ab} = 0$$

$$(\parallel) \quad \nabla_a \mathcal{T}^{ab} = 0$$

Stress associated with constraint acts as a source

It is separately conserved

- Unstretchable surface: $\Omega = 1$ Paper folding!

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